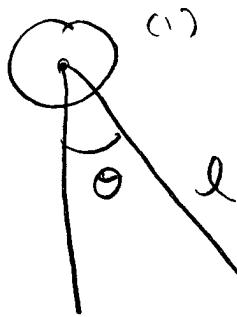


HW 3 Solutions

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KK 6.15



$$\tau = I \ddot{\theta} = -Mglsin\theta$$

for small θ , $\sin\theta \sim \theta$, and

$$\ddot{\theta} + \omega^2\theta = 0 \text{ with } \omega = \sqrt{\frac{Mgl}{I}}.$$

(2) The period is $T = \frac{2\pi}{\omega} = 2\pi \sqrt{\frac{I}{Mgl}}$

$\downarrow Mg$ The moment of inertia is

$$I = I_1 + I_2$$

$$= \left(\frac{1}{2}MR^2 \right) + \underbrace{\left(\frac{1}{2}MR^2 + ML^2 \right)}_{\text{parallel axis theorem}}$$

parallel axis theorem

$$I = M(R^2 + l^2)$$

KK 6.16

This is similar to the above, but now I is different b/c the first disk has been replaced by a pivot. So,

$$\omega = 2\pi \sqrt{\frac{Mgl}{I}} \text{ as before, but with } I = \frac{1}{2}MR^2 + Ml^2.$$

Since $T \propto \frac{1}{\omega}$, minimizing T corresponds to maximizing ω .

$$\omega \propto \frac{l}{\sqrt{\frac{R^2}{2} + l^2}}. \text{ The extrema satisfy } \frac{d\omega}{dl} = 0,$$

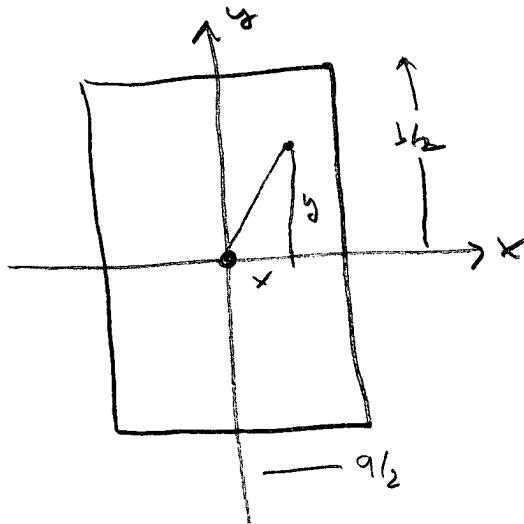
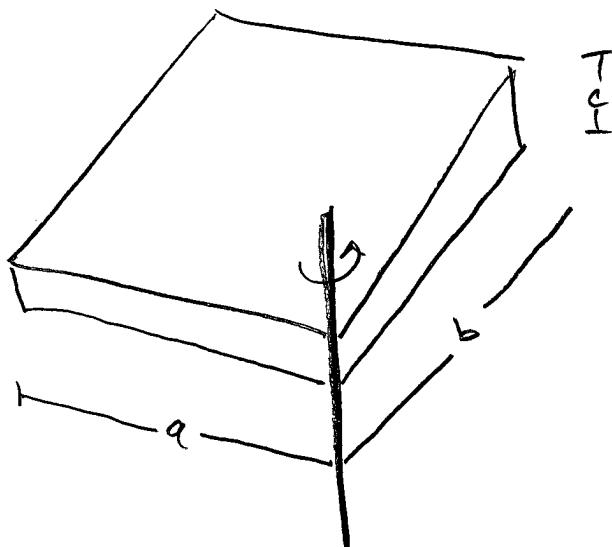
HKT 6.16 con't

$$\frac{d\omega}{dl} = \frac{d}{dl} \left(\frac{l}{R^2/2 + l^2} \right) = \frac{1}{R^2/2 + l^2} - \frac{l}{(R^2/2 + l^2)^2} \cdot 2l$$

$$= \frac{R^2/2 - l^2}{(R^2/2 + l^2)^2} = 0 \Rightarrow l = R/\sqrt{2}$$

HKT 12.6

First, calculate the moment of inertia about the center:



An arbitrary point on the large face is a distance $\sqrt{x^2+y^2}$ away from origin (axis of rotation). So,

$$I_z = \int_{-a/2}^{a/2} dx \int_{-b/2}^{b/2} dy \int_0^c dz (x^2+y^2) dm, \quad dm = \rho dV$$

$$= \frac{M}{abc} \int_0^{a/2} dx \int_0^{b/2} dy dz (x^2+y^2)$$

$$= \frac{M}{abc} \int_0^{a/2} dx \int_0^{b/2} dy (x^2+y^2)$$

$$= \frac{4M}{ab} \int_0^{a/2} dx \int_0^{b/2} dy (x^2+y^2)$$

HRK 12.6 con't

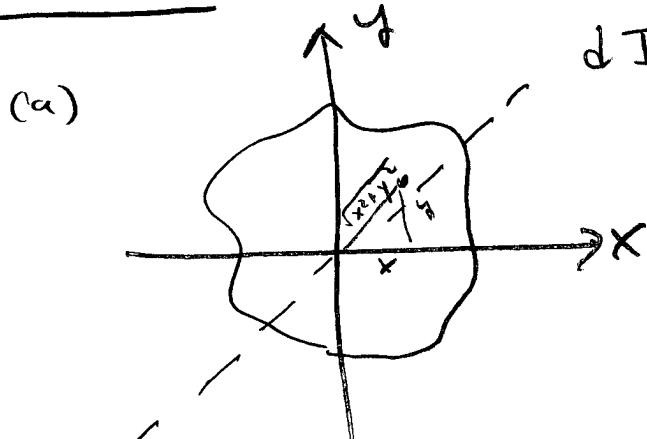
$$\begin{aligned}
 I_z &= \frac{4M}{ab} \left(\int_0^{b/2} dy \int_0^{a/2} dx x^2 + \int_0^{a/2} dx \int_0^{b/2} dy y^2 \right) \\
 &= \frac{4M}{ab} \left(\frac{b}{2} \frac{1}{3} \left(\frac{a}{2}\right)^3 + \frac{a}{2} \frac{1}{3} \left(\frac{b}{2}\right)^3 \right) = \frac{4M}{ab} \left(\frac{a^3 b}{3 \cdot 16} + \frac{a b^3}{3 \cdot 16} \right) \\
 &= \frac{M}{12} (a^2 + b^2)
 \end{aligned}$$

Now use the parallel axis theorem to translate axis to corner,

$$\begin{aligned}
 I_{\text{corner}} &= I_z + M \left(\left(\frac{a}{2}\right)^2 + \left(\frac{b}{2}\right)^2 \right) \\
 &= \frac{M}{12} (a^2 + b^2) + \frac{M}{4} (a^2 + b^2)
 \end{aligned}$$

$I = \frac{M}{3} (a^2 + b^2)$

HRK 12.9



$dI_x = dm y^2$

$dI_y = dm x^2$, and so

$dI_x + dI_y = dm(x^2 + y^2)$

$= dm r^2$

↑ distance to
z-axis

$= dI_z \sqrt{z^2}$

HRT 12.9 con't

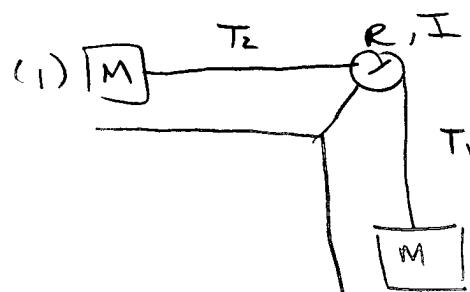
(b) for a circle, $dI_z = dm r^2 = dI_x + dI_y$, but $dI_x = dI_y$ by symmetry, so $\frac{dI_{\text{any diameter}}}{\text{diameter}} = \frac{1}{2} dI_z$.

$$I_{\text{any diameter}} = \frac{1}{2} I_z = \frac{1}{2} \left(\frac{M}{\pi R^2} \right) \int (dr d\phi) r^2 = \frac{1}{2} \left(\frac{M}{\pi R^2} \right) (2\pi) \frac{R^4}{4}$$

$$= \frac{MR^2}{4}$$

$I_{\text{any diameter}} = \frac{MR^2}{4} = \frac{1}{2} I_z$

HRT 12.29



$a = \text{const.}$, we are told.

no slip means $R\alpha = a$, so $\alpha = \frac{a}{R}$.

$$\begin{aligned} \text{So } \Theta &= \Theta_0 + \omega_0 t + \frac{1}{2} \alpha t^2 \\ &= \frac{1}{2} \frac{a}{R} t^2. \end{aligned}$$

$a = \frac{2R\Theta}{t^2}, \alpha = \frac{2\Theta}{t^2}$

Summing up the forces on block (2),

$$\sum F_y = ma = mg - T_1. \quad (2)$$

Also, the angular acceleration of the pulley is caused by a torque,

$$\tau = R(T_1 - T_2) = I\alpha, \text{ so}$$

$$\text{and } -\frac{I\alpha}{R} + T_1 = T_2, \text{ so}$$

$$\begin{aligned} T_1 &= m(g - a) \\ &= m\left(g - \frac{2R\Theta}{t^2}\right) \end{aligned}$$

$$T_2 = m\left(g - \frac{2R\Theta}{t^2}\right) - \frac{2I\Theta}{t^2 R}$$

$$T_2 = mg - \frac{2Rm\Theta}{t^2} - \frac{2I\Theta}{t^2 R} = mg - \frac{2\Theta}{t^2} \left[mR + \frac{I}{R} \right].$$

HRT 12.34

energy is conserved, so $E_i = E_f$.

E_i is purely potential energy, $E_i = mgh$

E_f is purely kinetic (choosing to measure our height from this point),

$$E_f = \frac{1}{2}mv^2 + \frac{1}{2}I_{\text{sphere shell}}\omega_{\text{sphere shell}}^2 + \frac{1}{2}I_{\text{pulley}}\omega_{\text{pulley}}^2$$

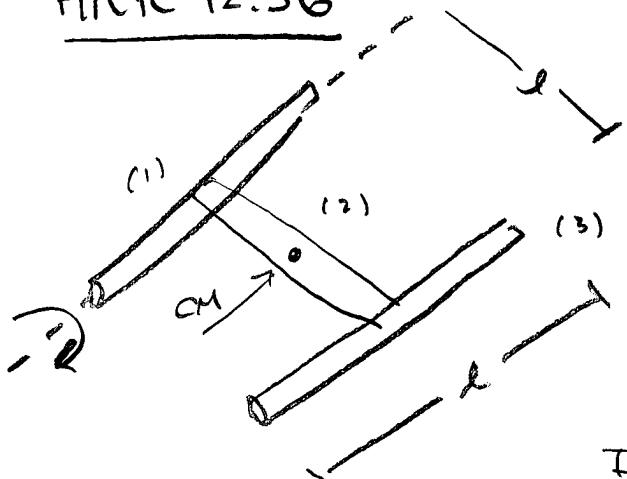
using $\omega_{\text{sphere shell}} = \frac{v}{R}$, $\omega_{\text{pulley}} = \frac{v}{r}$, and

$$I_{\text{sphere shell}} = \frac{2MR^2}{3}, \quad I_{\text{pulley}} = ?$$

we have $E_i - E_f = 0 \Rightarrow$

$$v = \sqrt{\frac{2mgh}{m + \frac{2}{3}M + \frac{IP}{r^2}}}$$

HRT 12.36



$$I = I_1 + I_2 + I_3$$

$I_1 = 0$ b/c The rod is aligned w/ The axis. Every point on the rod is rotating about the axis at a very small distance, which is zero in the limit where the rod is infinitely thin.

$$I_2 = \frac{1}{3}ML^2$$

$$I_3 = I_{CM} + ML^2 = \frac{4}{3}ML^2$$

HRT 12.36 con't

$$E_i = E_f \Rightarrow$$

$$\frac{1}{2} I \omega^2 = M_{\text{tot}} g \Delta h_{\text{cm}}, \quad M_{\text{tot}} = 3M, \quad \Delta h_{\text{cm}} = \frac{L}{2}, \text{ and}$$

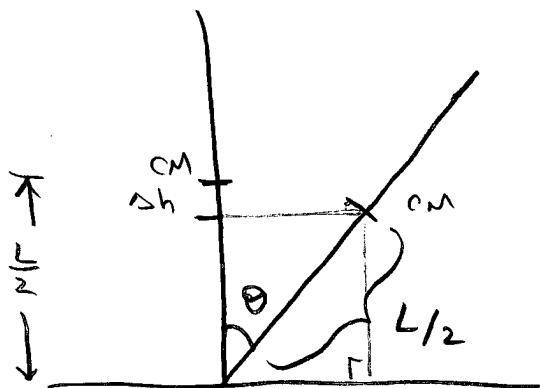
$$\omega^2 = \frac{2 \cdot 3 \cdot M \cdot g \cdot \frac{L}{2}}{\frac{4}{3} M L^2} = \frac{9g}{4L}, \text{ and}$$

$$\omega = \frac{3}{2} \sqrt{\frac{g}{L}}$$

HRT 12.38

$$\text{Energy conservation: } E_i = 0 = E_f = \frac{1}{2} I \omega^2 - mg \Delta h_{\text{cm}}$$

want Δh_{cm} in terms of θ :



$$\frac{L}{2} - \Delta h = \frac{L}{2} \cos \theta, \text{ so}$$

$$\Delta h = \frac{L}{2} (1 - \cos \theta).$$

so we can use this to find ω now:

$$\omega^2 = \frac{2mg \Delta h_{\text{cm}}}{I}$$

$$= \frac{mg L (1 - \cos \theta)}{I}$$

$$\text{here } I = \frac{1}{3} M L^2, \text{ so}$$

$$\omega^2 = \frac{3g}{L} (1 - \cos \theta)$$

HRT 12.38 con't

$$\alpha = \frac{d}{dt} \dot{\theta} = \frac{d}{dt} \omega = \frac{1}{2\omega} \frac{d}{dt} \omega^2 = \frac{1}{2\omega} \frac{3g}{L} \frac{d}{dt} (1 - \cos \theta)$$

$$\alpha = \frac{3g}{2L} \sin \theta.$$

We want to know

- (a) radial acceleration

$$a_r = L\omega^2 = 3g(1 - \cos \theta)$$

- (b) tangential acceleration,

$$a_t = L\dot{\theta} = \frac{3g}{2} \sin \theta$$

(c) if $a_t = g$, Then $\sin \theta = \frac{2}{3} \Rightarrow \theta \approx 0.73$ or 42°

- (d) The top end is falling faster than a freely falling body, which can only happen as a result of internal stresses which can break the chimney.

H&R 12.39 Let Earth be a uniform sphere,

$$T = \frac{2}{5} MR^2$$

$$KE = \frac{1}{2} I \omega^2 = \frac{1}{2} \frac{2MR^2}{5} \left(\frac{2\pi}{T}\right)^2$$

$$\begin{aligned} \frac{d(KE)}{dt} &= \frac{4\pi^2}{5} MR^2 \frac{d}{dt} T^{-2} \\ &= \frac{4\pi^2}{5} MR^2 \left(-\frac{2}{T^3}\right) \frac{dT}{dt} \quad . \text{ plugging in Nr #'s,} \end{aligned}$$

(a)

$$\boxed{\frac{dKE}{dt} = -1.88 \cdot 10^{12} \frac{\pi}{s}}$$

(b) average acceleration is

$$\alpha = \frac{d\omega}{dt} = 2\pi \frac{d}{dt} T^{-1} = -\frac{2\pi}{T^2} \frac{dT}{dt}$$

$$\boxed{\alpha = 2.67 \cdot 10^{-22} \frac{\text{rad}}{\text{s}^2}}$$

(c)

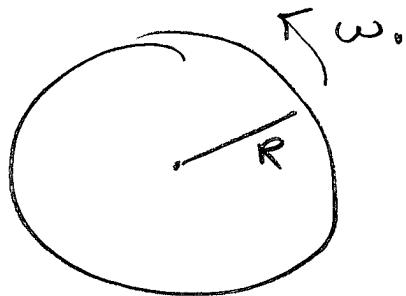
$$T = FR = I\alpha, \text{ so}$$

$$\boxed{F = \frac{I\alpha}{R} = 4.06 \cdot 10^9 N}$$

HRK 12.40

(a)

$$d\tau = -r \, dF$$



$dF = f \, dA$, f is the force density,

$$f = \frac{F}{A} = \frac{\mu N}{A} = \frac{\mu Mg}{\pi R^2},$$

$$\begin{aligned} \text{So, } \tau &= \int d\tau = - \int_0^{2\pi} d\phi \int_0^R dr r^2 f \\ &= -2\pi f \frac{R^3}{3} = -\frac{2}{3}\mu MgR \end{aligned}$$

(b)

$$\tau = I\alpha, \text{ so } \alpha = \frac{\tau}{I} \quad (\text{const}), \text{ So.}$$

$\omega = \omega_0 + \alpha t$. at $t = t_f$, let $\omega = 0$. Then

$$0 = \omega_0 + \alpha t_f, \text{ and } \dots$$

$$t_f = -\frac{\omega_0}{\alpha} = -\frac{\omega_0 I}{\tau} = \frac{-\omega_0 \left(\frac{1}{2}MR^2\right)}{-\frac{2}{3}\mu MgR}$$

$$t_f = \frac{3\omega_0 R}{2\mu g}$$