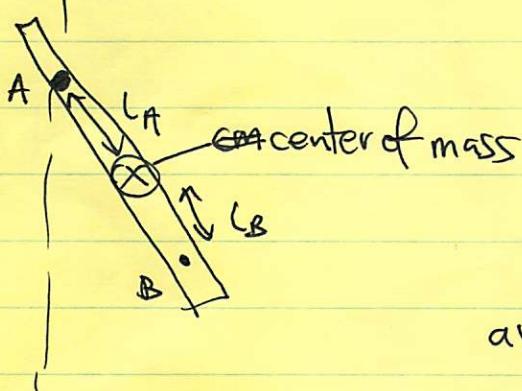


Week 3 - Pendulums +

Rotational & Translational Motion

If

Kater's Pendulum



Previous 16th - 20th Century
most accurate measurement
of g with pendulums.

$$T = 2\pi \sqrt{\frac{k^2 + l^2}{gL}}$$

average over
many periods

limited by accuracy to which k (finite moment
of inertia of weight)
& L (due to uncertainty in c.m. of pendulum).

Kater's pendulum eliminates this uncertainty!

We just need Pendulum can be suspended for.
2 adjustable edges, with period.

$$T_A = 2\pi \left(\frac{k^2 + l_A^2}{gL_A} \right)^{1/2}$$

$$T_B = 2\pi \left(\frac{k^2 + l_B^2}{gL_B} \right)^{1/2}$$

Adjust l_A, l_B until $T_A = T_B = T$.

(1g)

Then:

$$\frac{k^2 + l_A^2}{g l_A} = \frac{k^2 + l_B^2}{g l_B}$$

$$\Rightarrow l_B(k^2 + l_A^2) = l_A(k^2 + l_B^2)$$

~~$$k^2 = l_B^2 - l_A^2 = (l_B - l_A)(l_B + l_A)$$~~

$$\Rightarrow k^2(l_B - l_A) = l_B l_A \quad \cancel{\approx l_B + l_A} \\ (l_B - l_A)$$

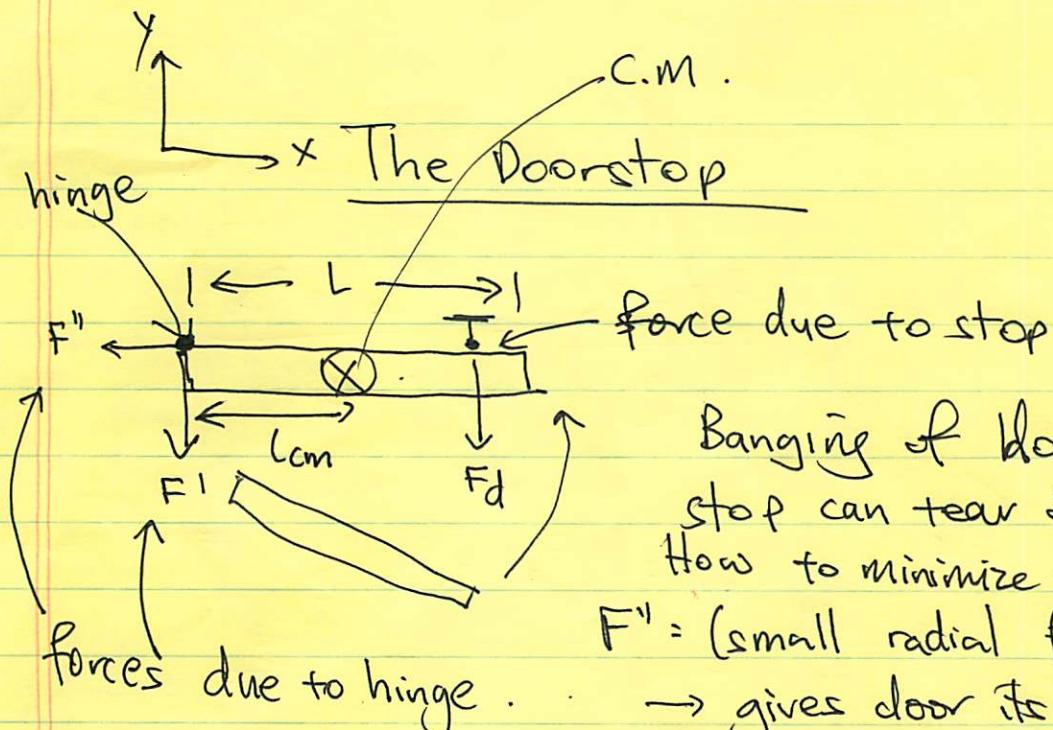
$$\Rightarrow k^2 = l_B l_A$$

$$\Rightarrow T = 2\pi \left(\frac{k^2 + l_A^2}{g l_A} \right)^{1/2}$$

$$= 2\pi \left(\frac{l_A(l_B + l_A)}{g l_A} \right)^{1/2} = 2\pi \left(\frac{l_B + l_A}{g} \right)^{1/2}$$

Only geometrical quantity is $l_A + l_B$,
 which can be measured with great precision!
 Don't need to know c.m. ...

1h



Banging of door against stop can tear off hinge.
How to minimize this?

F'' = (small radial force)

→ gives door its centripetal acceleration.

~~F' , F_d~~ = large forces which bring door to rest.

Forces on hinge equal & opposite to F' , F''

try to minimize this!

~~Goal~~ To get expression for F' , consider

- angular momentum of door about hinge
- linear momentum of c.m.

$$\tau = \frac{dL}{dt} \Rightarrow dL = \tau dt$$

$$\Rightarrow \Delta L = \int_{t_i}^{t_f} \tau dt$$

~~I_{ω_0}~~

$$I_{\omega_0} = \int F_d dt$$

integral is over duration of (1) collision.

(1i)

Similarly $F = \frac{dp}{dt}$

$$\Rightarrow \Delta p = \int F dt$$

$$P_f = 0$$

$$P_i = mv_y = m l_{cm} \omega_0$$

$$F_y = -(F^I + F_d)$$

$$P_f - P_i = +m l_{cm} \omega_0 = + \int (F^I + F_d) dt$$

But from (1),

$$\int F_d dt = \frac{I \omega_0}{L}$$

$$\Rightarrow Ml_{cm} \omega_0 - \frac{I \omega_0}{L} = \int F^I dt$$

$$\Rightarrow \int F^I dt = \omega_0 \left(Ml_{cm} - \frac{I}{L} \right)$$

Choose the doorstop to be at a distance

$$L = \frac{I}{Ml_{cm}} \rightarrow \text{can make impact force zero!}$$

for uniform door of width ω ,

$$I = \frac{M\omega^2}{3} \quad (\text{just like for stick})$$

Tj

$$l_{cm} = \frac{w}{2}$$

$$\Rightarrow l = \frac{\cancel{M} I}{\cancel{M} l_{cm}} = \frac{\cancel{M} \frac{w^2}{3}}{\cancel{M} \frac{w}{2}}$$

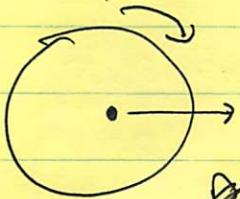
$$= \frac{2}{3} w$$

Door stop must be at ^{height of} c-m. rather than at floor level → otherwise force on 2 hinges are unequal.

Dist. l is center of percussion

Just like hitting a sweet spot on a baseball bat
 avoid minute impact

Combined Rotational & Translational Motion



Now relax assumption of pure rotation
— consider rolling drum.

What is the appropriate axis to use?

Recall Chasles's theorem: any motion can be decomposed into translation of c.m. & rotation about c.m.



So we will use c.m. coordinates!

fixed axis rotation:

Only consider motion where axis of rotation doesn't change direction in c.m. frame \rightarrow call it the z axis.

Show that

\leftarrow rotation of body about center of mass

$$\mathbf{L}_z = I_{\text{cm}} \omega + (\mathbf{R} \times \mathbf{M}\mathbf{V})_z$$



↑
motion of c.m. wrt origin
of ~~inertial~~ coordinate system

"spin angular momentum"

↑
"Orbital angular momentum"

E.g. think of earth's spin + orbital

E.g. think of earth's spin + orbital motion.

Will also encounter this in atomic physics

\rightarrow spin & orbital \vec{L} of electron.

Proof:

$$\vec{L} = \sum_{j=1}^N (\vec{r}_j \times m_j \vec{v}_j) = \sum_j (\vec{r}_j \times m_j \vec{v}_j)$$

(all wrt inertial coordinate system)

center of mass \vec{R} is described by a position vector:

$$\vec{R} = \frac{\sum m_j \vec{r}_j}{M}$$

Now use c.m. coordinates \vec{r}'_j :

$$\vec{r}_j = \vec{R} + \vec{r}'_j$$

Then

$$\vec{L} = \sum_j (\vec{r}'_j + \vec{R}) \times m_j \vec{v}'_j$$

$$= \sum_j (\vec{R} + \vec{r}'_j) \times m_j (\vec{R} + \vec{r}'_j)$$

$$= \vec{R} \times \sum_j m_j \vec{R} + \vec{R} \times \sum_j m_j \vec{r}'_j$$

~~$\vec{R} \times \sum_j m_j \vec{r}'_j$~~

$$+ \sum_j m_j \vec{r}'_j \times \vec{R} + \sum_j m_j \vec{r}'_j \times \vec{r}'_j$$

Consider ①

$$\vec{R} \times \sum_j m_j \vec{R} = \vec{R} \times M \vec{R} = \vec{R} \times M \vec{V}$$

velocity of
c.m. wrt
inertial
system.

3-3

~~$$\textcircled{2} \quad \vec{R} \times \sum m_j \vec{r}_j + \textcircled{4} \quad \sum m_j \vec{r}_j \times \vec{R}$$~~

For terms $\textcircled{2}$ & $\textcircled{4}$, consider

$$\begin{aligned}\sum m_j \vec{r}_j' &= \sum m_j (\vec{r}_j - \vec{R}) \\ &= M \vec{R} - M \vec{R} \\ &= 0.\end{aligned}$$

Also $\sum m_j \vec{r}_j'' = 0$ (since term is always zero,
so $\#$ it's a const and
 $\textcircled{2}$ & $\textcircled{4} = 0$. time derivative is zero).

Now we have:

$$\vec{L} = \vec{R} \times M \vec{V} + \sum \vec{r}_j' \times m_j \vec{r}_j''$$

angular momentum due
to c.m. motion

angular momentum
due to motion
about c.m.

~~Since~~ We are only considering fixed axis ROTATION!
rotation (will consider arbitrary axis of rotation -
e.g., gyroscopes, later).

Take z const

$$L_z = (\vec{R} \times M \vec{V})_z + (\sum \vec{r}_j' \times m_j \vec{r}_j'')_z$$

These terms are all evaluated in the c.m.
frame.

So it's identical to the case of pure rotation we've already analysed!



$$(\sum m_j \vec{r}_j' \times \dot{\vec{r}}_j')_z = \sum (m_j \vec{s}_j \times \vec{s}_j) \quad \text{or} \quad \vec{s}_j' = \vec{s}_j \omega$$

$$= \sum m_j s_j^2 \omega = I_o \omega$$

$$L_z = I_o \omega + (R \times M \vec{V})_z$$

angular momentum
about c.m.

angular momentum about
origin

Q.E.D!

Two things to note:

→ this expression still valid if the c.m. is accelerating, since \vec{L} is calculated wrt inertial coord. system.

→ spin angular momentum is independent of coord system (no change of coord system can eliminate spin)

but orbital angular momentum disappears if ~~the~~ origin is c.m.

Week 3

Rotation + Translation (Cont.)

①

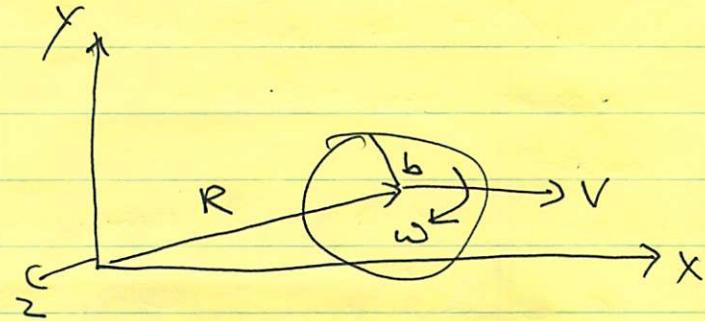
Angular Momentum of a Rolling Wheel

Calculate angular momentum of a rolling wheel of mass M , radius b which rolls uniformly & w/out slipping.

about center of mass $\rightarrow I_0 = \frac{1}{2} M b^2$

$$L_0 = -I_0 \omega \hat{z}$$

$$= -\frac{1}{2} M b^2 \omega \hat{z}$$



$$(R \times M V)_z = \cancel{M b V} \quad (\text{since } R \perp)$$

$$= \cancel{R_y}$$

$$\approx \cancel{\Delta t}$$

$$= -M R_y V_x$$

$$= -M b V$$

$$L_z = I_0 \omega + (\vec{R} \times \vec{M V})_z$$

$$= -\frac{1}{2} M b^2 \omega - \cancel{M b V}$$

$$\cancel{\neq} \Rightarrow M b(b \omega)$$

$$= -\frac{3}{2} M b^2 \omega$$

$$V = b \omega$$

if noslip

(2)

Torque

$$\tau = \sum r_j \times f_j$$

Torque also divides into 2 components :

$$\tau = \sum \vec{r}_j \times \vec{f}_j$$

$$= \sum (\vec{r}_j^1 + \vec{R}) \times \vec{f}_j$$

$$= \sum (r_j^1 \times \vec{f}_j) + \cancel{\sum R \times \vec{f}_j}$$

~~torque~~ \rightarrow
torque about center of mass

$$\vec{R} \times \vec{F} (= \sum \vec{f}_j)$$

\uparrow
torque due to
total external
force acting
at c.m.

For fixed axis rotation, $\omega = \omega \hat{k}$

$$\Rightarrow \tau = \tau_o + (R \times F)_z$$

But from $\cancel{\text{if}} L_z = I_o \omega + (\vec{R} \times M \vec{V})_z$,

$$\frac{dL_z}{dt} = I_o \frac{d\omega}{dt} + \cancel{\frac{d}{dt}(\vec{R} \times M \vec{V}_z)}$$

$$= I_o \alpha + (\vec{R} \times M \vec{a})_z$$

$$\Rightarrow \tau_o + (\vec{R} \times \vec{F}_o)_z = I_o \alpha + (R \times M \vec{a})_z$$

$$= I_o \alpha + (R \times \vec{F})_z$$

(using $\vec{F} = M \vec{a}$)

~~cancel~~

$$\Rightarrow \boxed{\tau_o = I_o \alpha}$$

(3)

i.e. $\tau_o = I\alpha$ is true even if c.m. is accelerating; rotational motion about c.m. depends only on torque about c.m., independent of translational motion.

~~Final dev~~ Finally, let's look at kinetic energy:

$$K = \frac{1}{2} \sum m_j v_j^2$$

$$= \frac{1}{2} \sum m_j (\vec{\omega}_j + \vec{V})^2$$

↑ motion about c.m.
↑ motion of c.m.

$$\begin{aligned} &= \frac{1}{2} \underbrace{\sum m_j \vec{\omega}_j^2}_{I} \omega^2 + \underbrace{\sum m_j \vec{\omega}_j \cdot \vec{V}}_{\text{since } \vec{r}_cm = r'_cm = \vec{0}} + \frac{1}{2} \sum m_j V^2 \\ &= \frac{1}{2} I_o \omega^2 + \frac{1}{2} M V^2 \end{aligned}$$

↑ kinetic energy of spin ↑ orbital center of mass motion

Summary: Formulae for Fixed Axis Motion

Pure rotation = no translation

$$L = I\omega$$

$$\tau = I\alpha$$

$$K = \frac{1}{2} I\omega^2$$

Rotation & translation (subscript 0 refers to c.m.)

$$L_z = I_o \omega + (R \times M V)_z$$

(4)

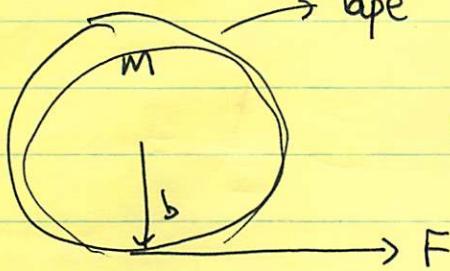
$$\tau_2 = \tau_0 + (\vec{R} \times \vec{F})_L$$

$$\tau_0 = I_0 \alpha$$

$$K = \frac{1}{2} I_0 \omega^2 + \frac{1}{2} M V^2.$$

We are now just going to work a bunch of examples

Disk on Ice



Pull by const force by
tape wound about circumference.

Disk slides on ice w/out friction.
What is its motion?

(this depends on the
origin that we choose).

We can analyse the torque & angular momentum
using different origins. ~~that~~

→ they will be origin dependent.

→ but a, α are not frame dependent

~~Ace~~
$$a = \frac{F}{M}$$
 (in all frames).

Analyze in frame of center of mass.

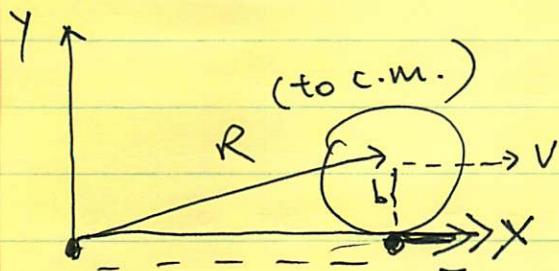
$$\tau_0 = b F$$

$$= I_0 \alpha$$

$$\Rightarrow \boxed{\alpha = \frac{b F}{I_0}}$$

(5)

Let's check that we get the same answer in another frame



$$\begin{aligned} \vec{T}_z &= \vec{T}_o + (\vec{R} \times \vec{F})_z \\ &= b\vec{F} \cancel{\times b} = 0 \end{aligned}$$

As expected, since

\vec{F} is parallel to \vec{R}

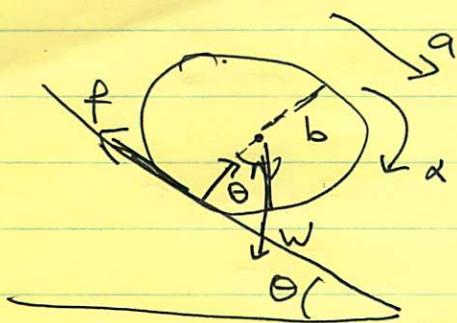
Hence, \vec{L} is conserved.

$$\begin{aligned} L_z &= I_o \omega + (\vec{R} \times M\vec{V})_z \\ &= I_o \omega - bMV \end{aligned}$$

$$\frac{dL_z}{dt} = I_o \alpha - bMa = 0$$

$$\Rightarrow \alpha = \frac{bMa}{I_o} = \frac{bF}{I_o} \quad (\text{as before})$$

Drum Rolling down a plane



Uniform drum, rolls w/out slipping down an inclined plane

$$I_o = \frac{M b^2}{2}$$

Want to solve for acceleration along plane.

Again, solve 2 ways ~~not~~, using different coord system.

(6)

Trans Motion of c.m. along plane:

$$W \sin \theta - f = Ma.$$

(1)

Rotation about c.m.

$$bf = I_o \alpha \Rightarrow f = \frac{I_o \alpha}{b} \quad (2)$$

For rolling w/out slipping,
 $\cancel{\Rightarrow} a = b\alpha$.

(3)

~~Eliminate~~ Plug (2) into (1),

$$W \sin \theta - \frac{I_o \alpha}{b} = Ma$$

$$\Rightarrow W \sin \theta - \frac{I_o \alpha}{b} \left(\frac{a}{b} \right) \left(\frac{Mb^2}{2} \right) = Ma$$

$$\Rightarrow Mg \sin \theta - \frac{Ma}{2} = Ma$$

$$\Rightarrow \boxed{a = \frac{2}{3} g \sin \theta}$$

(N.B. if no rotation, would
 be ~~a~~)

\exists more complicated way of doing it,
 DEFER! $a = g \sin \theta$)